

Numerical Analyses and Sensitivity Studies for Development of the Pendulous Method

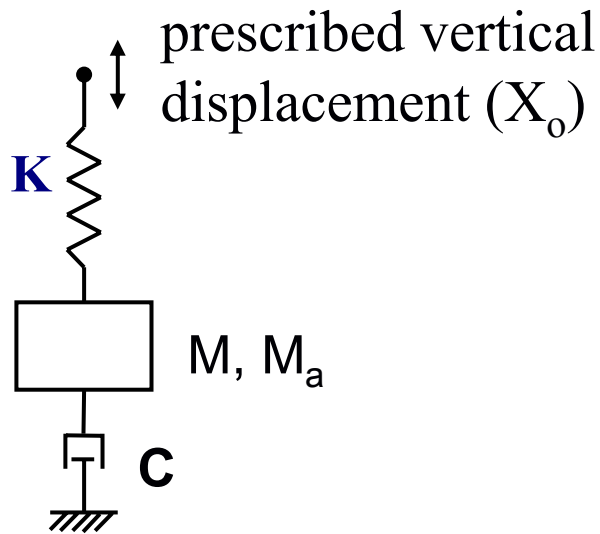
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Challenge of deployment in increasing WD (250 Te payload, 2000-3000 m WD)

- Disadvantage of wire rope: self weight + axial resonance (DAF)
- Synthetic fiber rope → issues to be solved: bending and heating + axial resonance (w/o heave compensation)



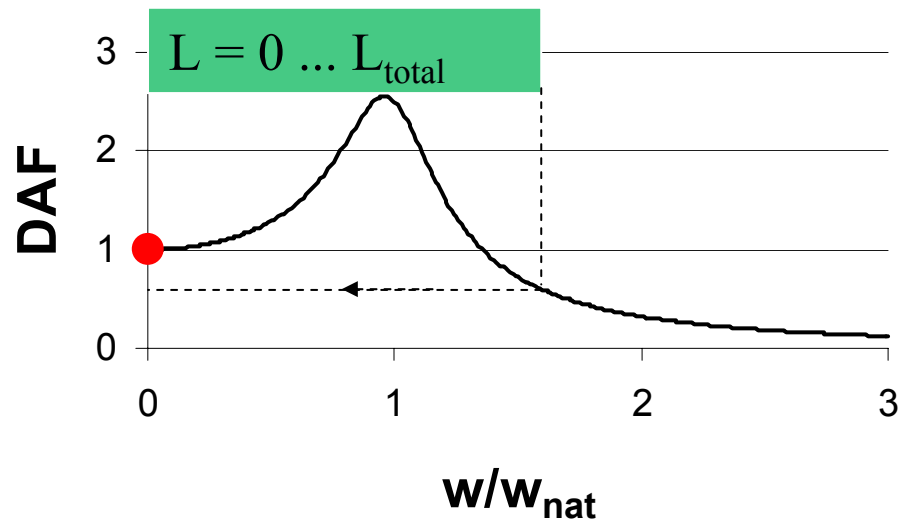


$$K = \frac{(EA)_{\text{equiv}}}{L}$$

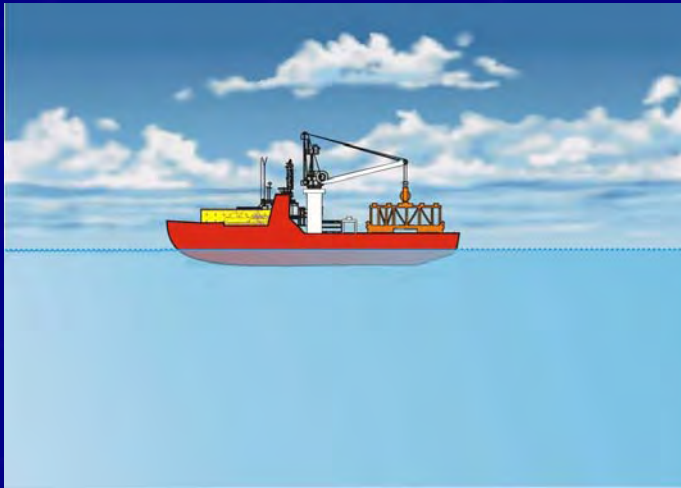
$$T_{\text{nat}} = \frac{2\pi}{\sqrt{\frac{K}{M + M_a + \frac{m}{e}}}} = \frac{2\pi}{w_{\text{nat}}}$$

frequency ratio β

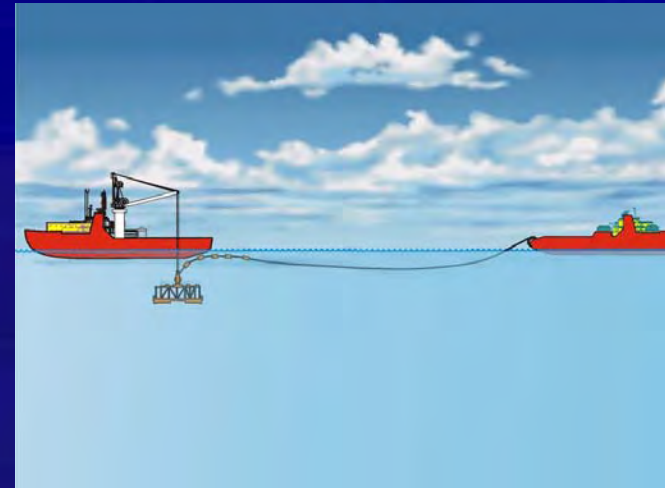
$$\frac{w}{w_{\text{nat}}} = f(L_{\text{deployed}})$$



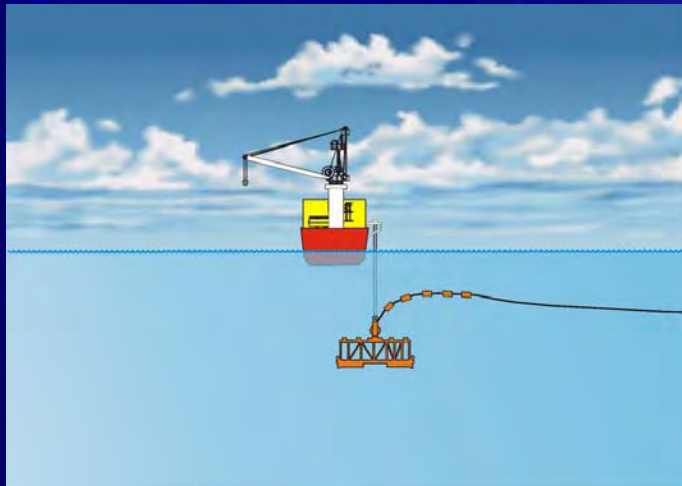
The Pendulous Method



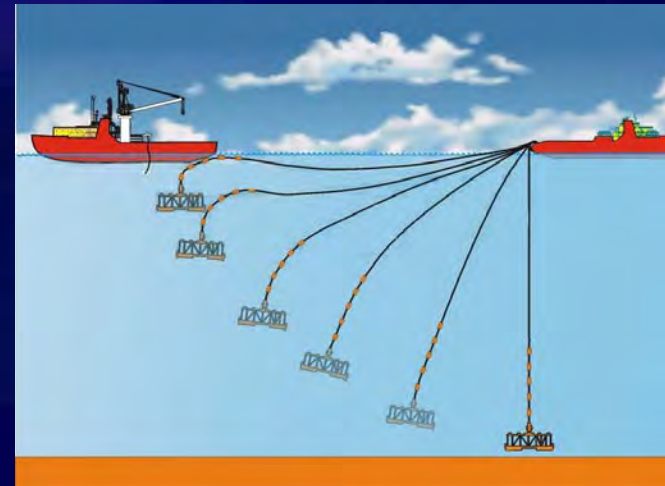
Transportation vessel



Overboarding



Hangoff



Pendulous Motion

The Pendulous Method (cont.)

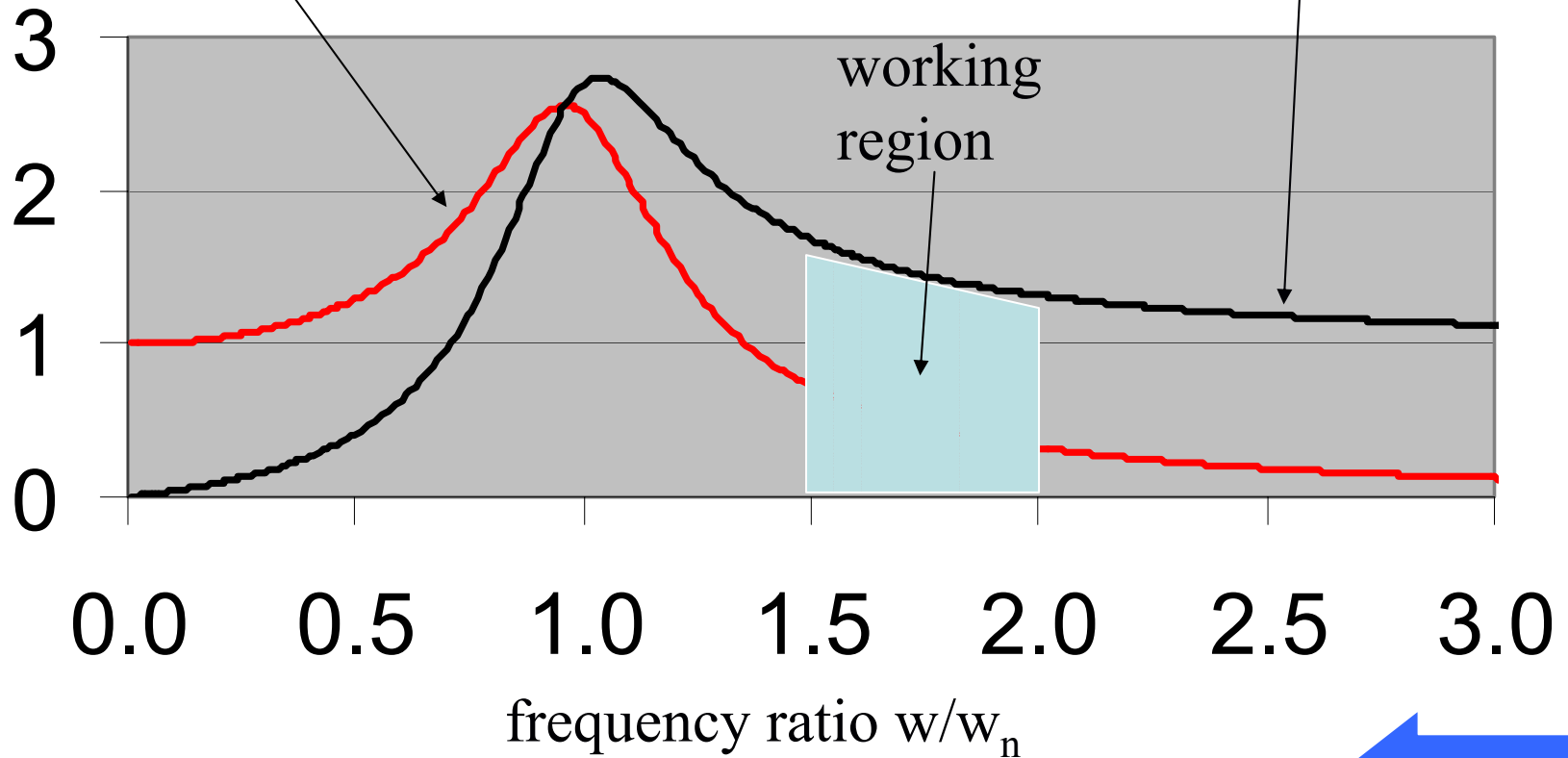
- Conceived to overcome the above constraints (DAF1)
- Utilization of the Pendulous Motion
- Utilization of two workboats
- Distance between vessels 90% of cable length
- Installation cable, from subsea hardware: wire rope with DBM, polyester and chain
- Due to drag the pendulous motion will be very slow



damping ratio $\xi = 0.20$

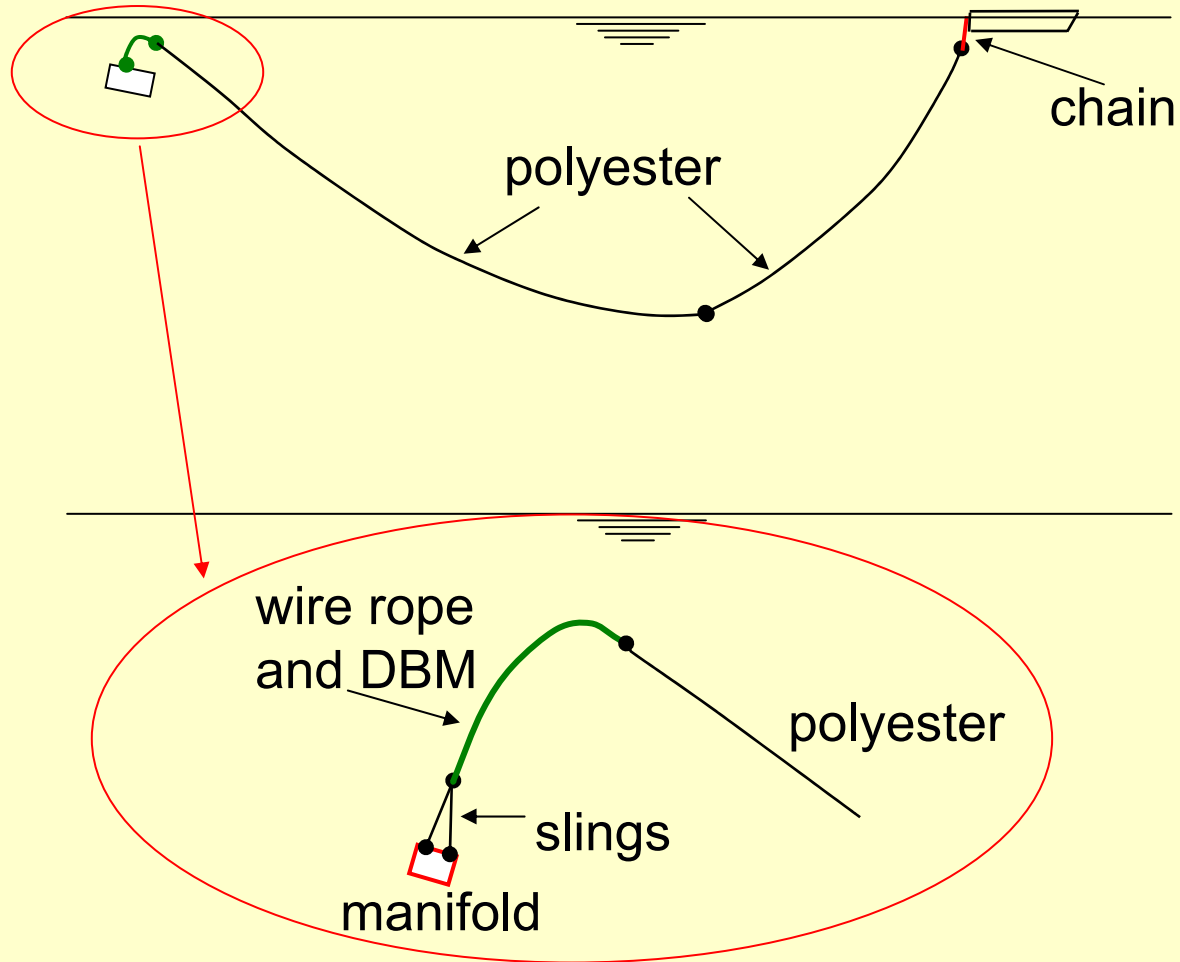
Amplitude of dynamic force (KX_0 multiplier)

DAF (displacements)



General system configuration

(side view, just after release)



System components

material	MBL (Te)	Diam. (mm)	length (m)	mass (kg/m)	wet wgt (kgf/m)	EA (MN)
chain R4	1097	105	30	222.7	193.8	840
polyester	1250	210	1600	29	7.40	300
wire rope	1000	127	60	57.8	44.7	965

Weight in air: 280 Te

WD: 1900m

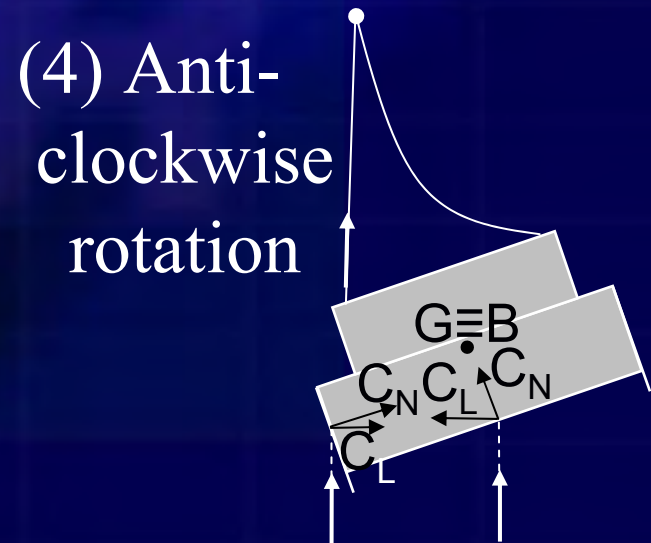
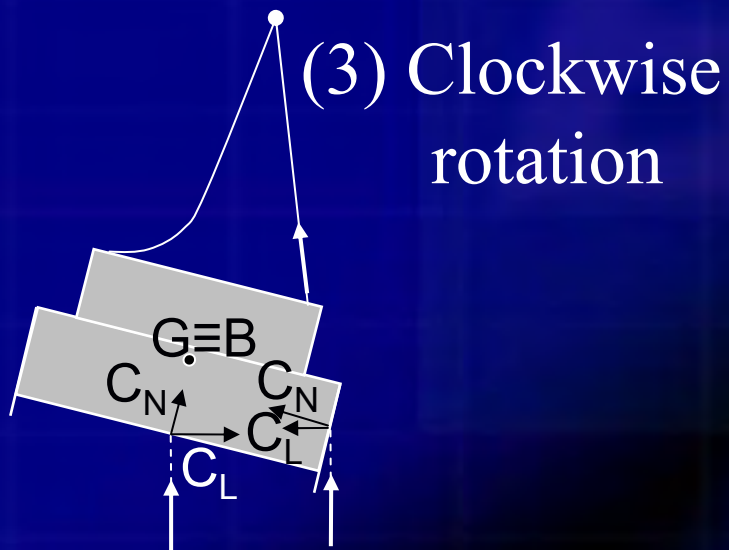
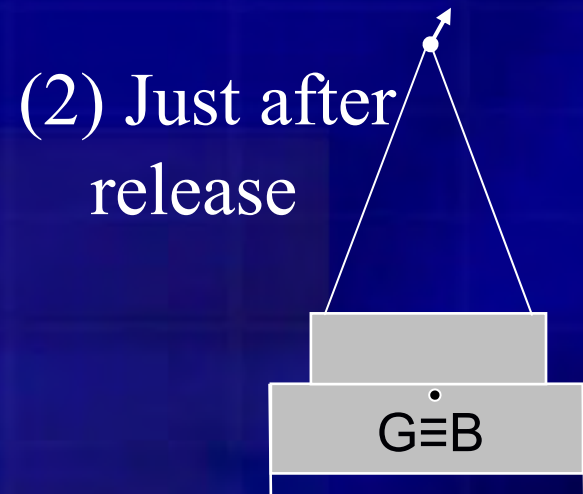
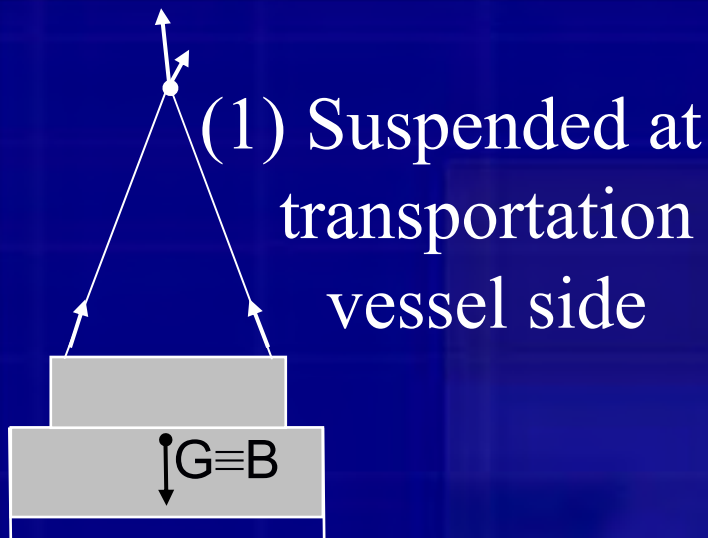
Dimensions: 16.6 x 8.5 x 5.1 m (L x B X H)

CG 3.15 m above base line (CG \equiv CB)

Physics of the problem

- Equipment of complex topology
- Volume of the envelope dimensions: 728 m^3
- Steel volume: 35.7 m^3 ($< 5\%$ total volume)
- Some assumptions are needed in order to simplify the computer model
- 1st approach to concentrate drag and added mass at CG
→ inadequate
- improvement → center of pressure and spatial distribution of drag and lift forces

Physics of the problem (cont.)



Development of the concept

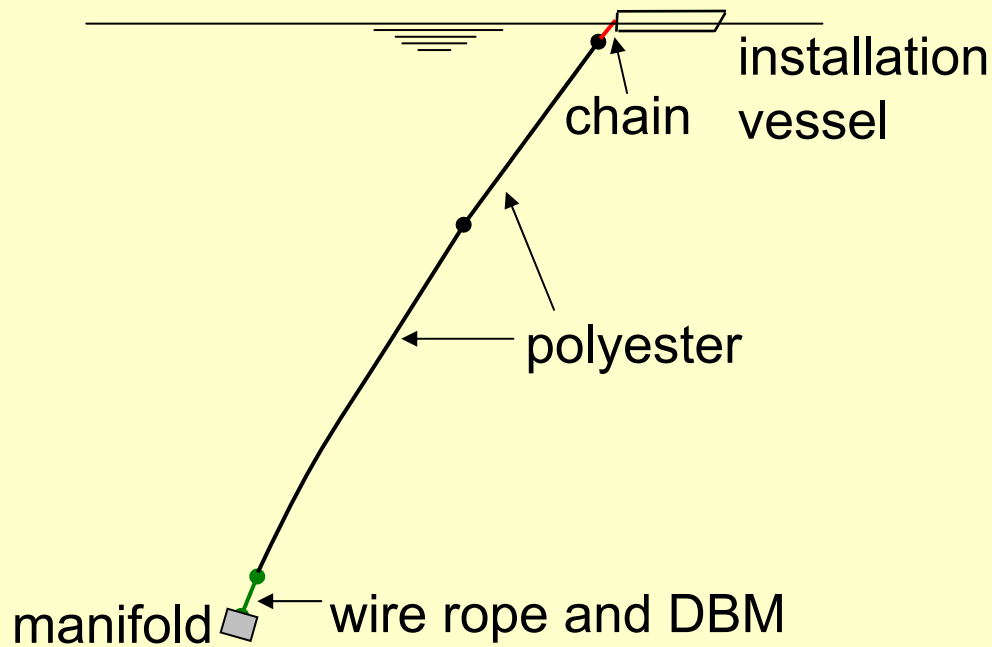
- Conceived in 2003, based on the procedure for installation of torpedo pile
- Numerical analyses with Orcaflex to demonstrate the feasibility – parametric study of main variables
- Model tests at LabOceano (UFRJ) in 2004
- 1:1 scale prototype test in December 2005

Some results of numerical analysis

➤ Three distinct phases:

- equipment hung off at side of transportation vessel
- pendulous motion
- equipment supported by installation vessel

Configuration 10 minutes after release

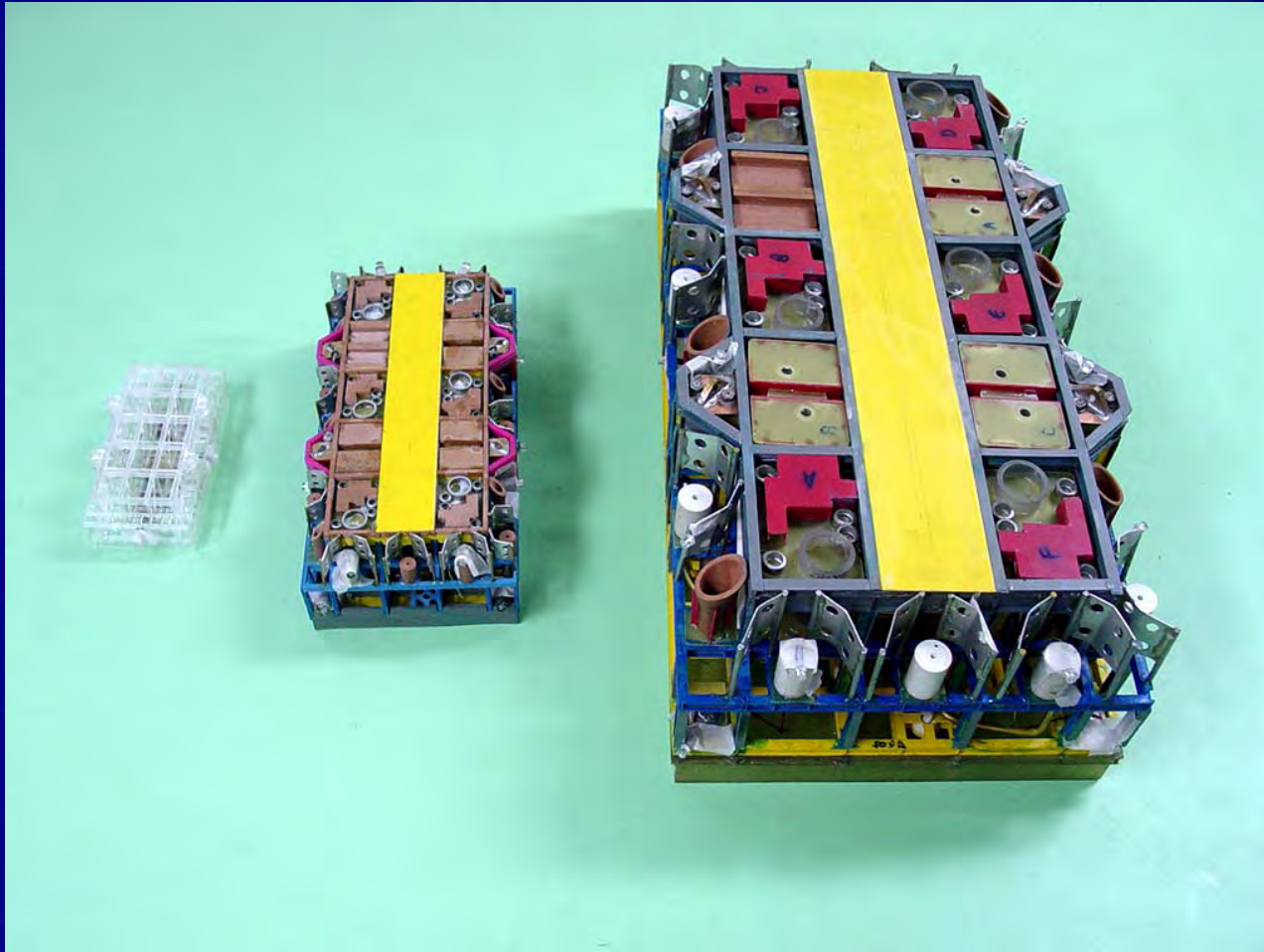


[1000 sec.avi](#)

[60 sec.avi](#)

Model tests

- Model tests at 1:35, 1:70 and 1:130 scales for manifold #2 – excessive rotations detected in some cases



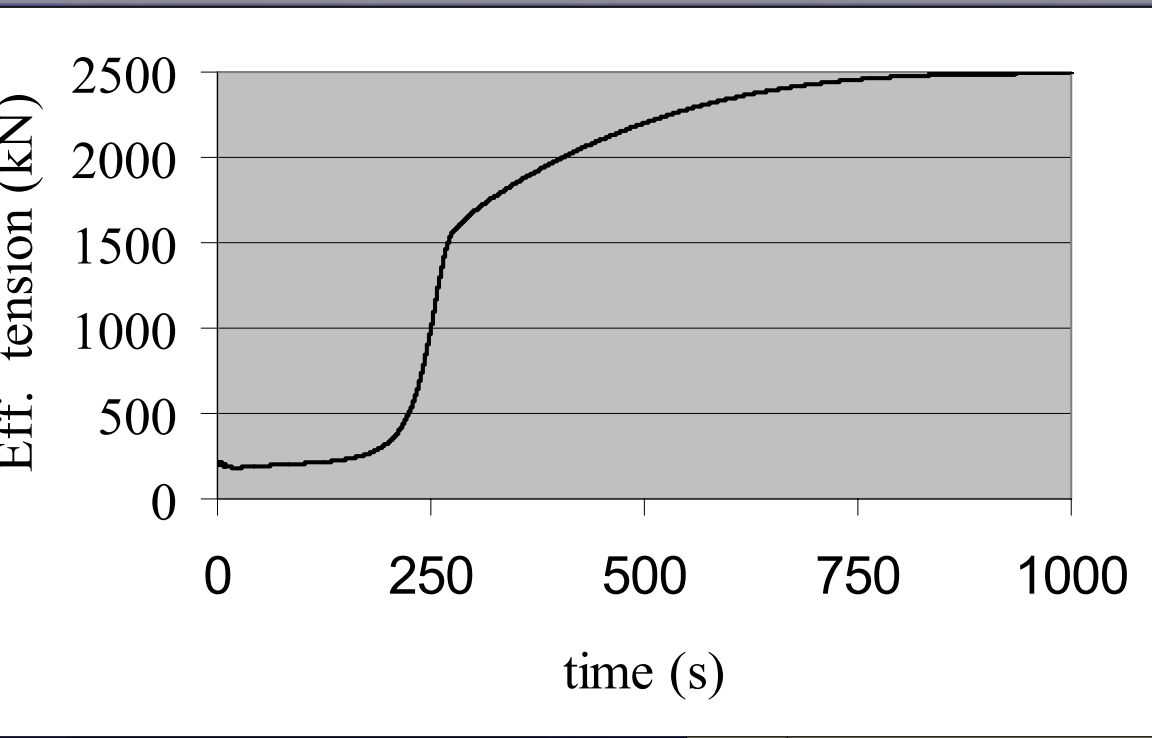
Model test

1:1 Prototype test

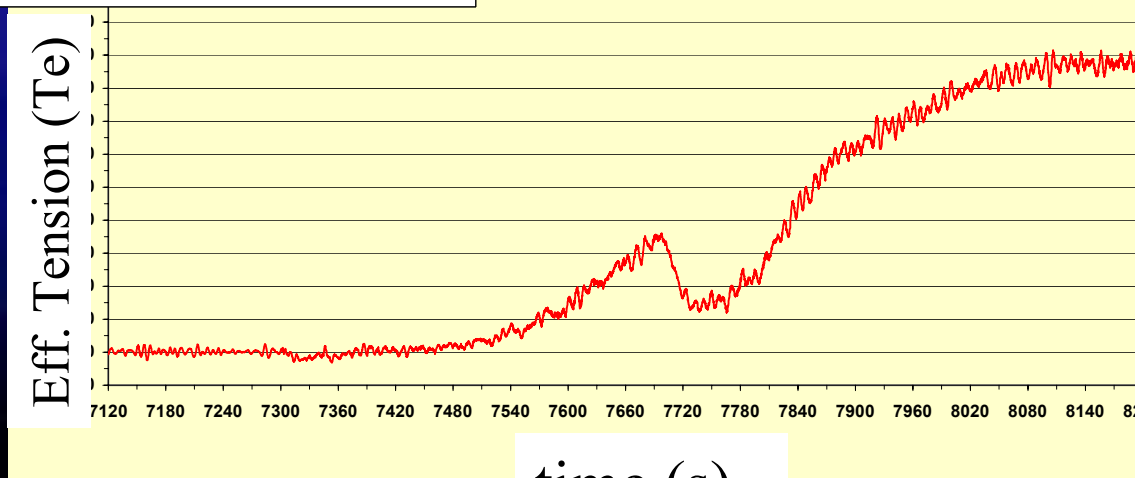
- Decision to build and install a 1:1 prototype for qualification of the method and installation procedure



Cable effective tension, installation vessel side



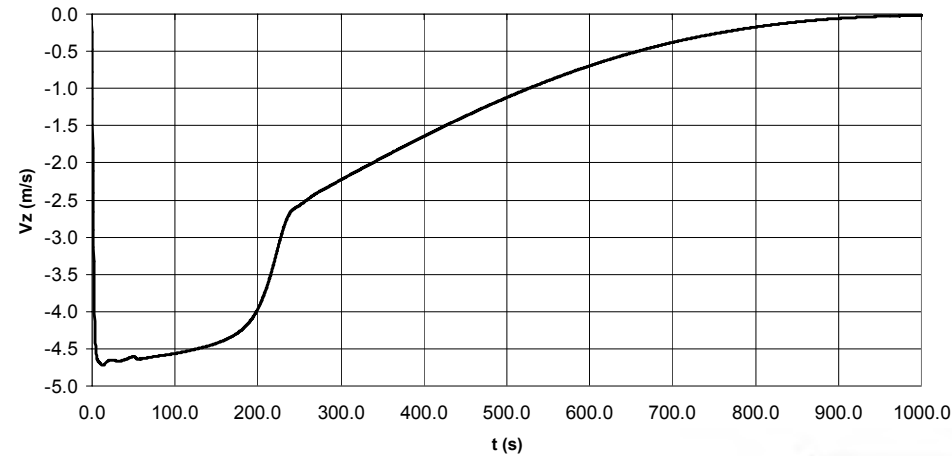
numerical analysis



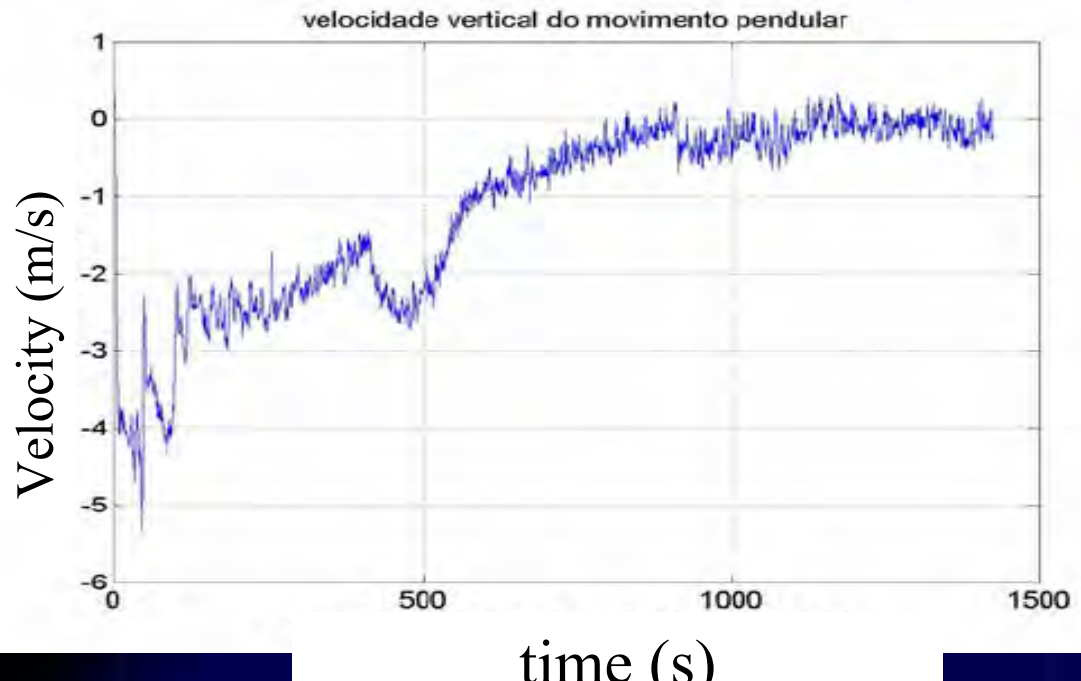
measurements

Manifold vertical velocity

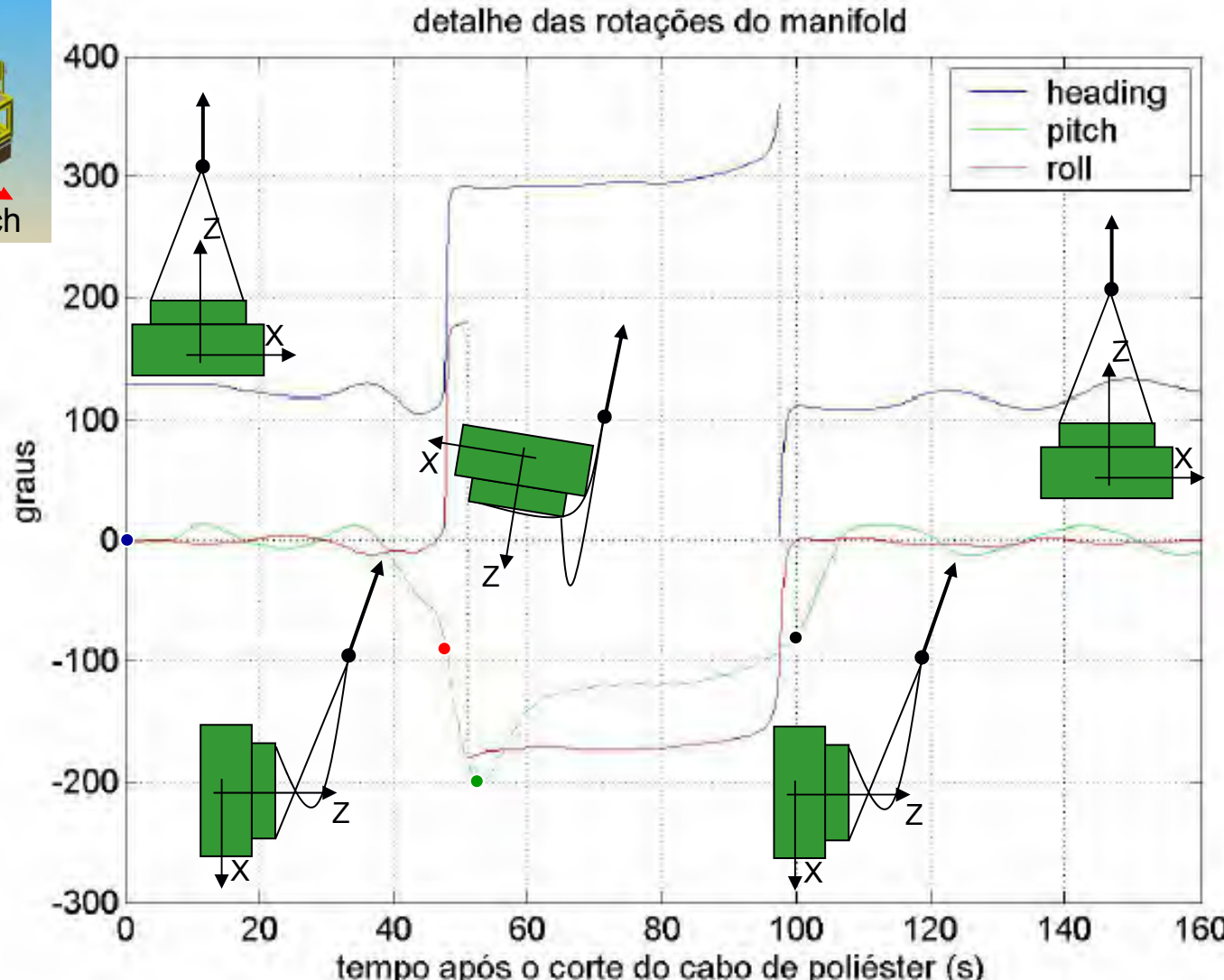
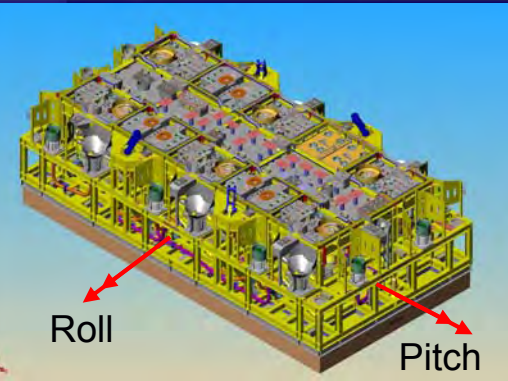
numerical analysis



measurements



Rotations



Excessive rotations and mitigation

➤ Identification of excessive rotations

➤ Mitigation

- improvement of hydrodynamic stability
 - ✓ a more adequate equipment geometry, e.g., near vertical (slightly slanted) panels around it
 - ✓ lowering of CG, e.g., additional weight at the equipment bottom or additional buoyancy at top
- increase of sling reaction forces at beginning
 - ✓ additional buoyancy to the distributed buoyancy modules
 - ✓ counterweight and wire rope

- Feasibility of the Pendulous Installation Method
 - Good comparison between numerical analysis x model tests x full scale test
 - Axial loads increase monotonically
 - No swing back and forth, due to damping
 - Attenuation of axial force, prevents resonance
- Control of rotations at beginning
 - counterweight + wire rope (MSG L 2&3)
 - improvement of hardware geometry (future)



27 12 2005